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## TWO-ARM BELT TENSIONER

TECHNICAL FIELD

The present invention relates to a two-arm belt  
10 tensioner for an internal combustion engine belt drive.

BACKGROUND ART

As is known, in engine belt drives, an endless  
belt connects the drive shaft to an electric machine  
and possibly one or more auxiliary members. When using  
a reversible electric machine, i.e. capable of  
15 operating both as a motor and generator, in which case,  
the belt branches have different tensions, depending on  
the operating mode of the electric machine, belt  
tension is controlled using two-arm belt tensioners.

The most commonly used two-arm belt tensioners  
20 comprise two belt tensioning arms, which have  
respective hinge portions mounted to rotate on the same  
hinge pin about a common axis, and support respective  
idle pulleys, each acting on a respective branch of the  
belt. The two arms are loaded towards each other by a  
25 normally spiral forcing spring located between the  
hinge portions of the arms to force the idle pulleys  
against the respective belt branches and so compensate  
for variations in belt tension.

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Though widely used, known belt tensioners of the type described above have the drawback of being extremely bulky, particularly in directions perpendicular to the hinge axis, and therefore unsuitable for highly compact combustion engines. This is mainly due to the type of forcing spring used, the size of which is determined by the spring having to provide sufficient force to ensure correct tensioning of the belt, while at the same time being flexible enough to cater to continual variations in the tension of the belt branches.

DISCLOSURE OF INVENTION

It is an object of the present invention to provide a two-arm belt tensioner designed to provide a straightforward, low-cost solution to the above problem, and in particular a two-arm belt tensioner which is highly compact, reliable and efficient, and is cheap and easy to produce.

According to the present invention, there is provided a two-arm belt tensioner for a belt drive of an internal combustion engine; the belt tensioner comprising a tubular supporting portion fixed to a fixed supporting structure; a first and a second arm fitted to said tubular supporting portion to rotate about a common hinge axis; a first and a second idle wheel fitted to respective ends of said first and said second arm and cooperating with respective branches of a belt of said drive; and elastic forcing means for forcing said first and said second arm towards each

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other to keep said wheels in contact with said respective branches of the belt; characterized in that said elastic forcing means comprise a torsionally elastic elongated member extending through said tubular supporting portion, coaxially with the hinge axis, and having respective opposite end portions projecting outwards of said tubular supporting portion; a first and a second end cap located at opposite axial ends of said tubular supporting portion, and each fitted in angularly fixed manner to a respective said end portion; said second cap being fitted integrally with one end of said second arm, and defining a radial opening fitted through with said first arm; angular connecting means being interposed between said first arm and said first cap, and housed inside said tubular supporting portion and said first and said second cap.

BRIEF DESCRIPTION OF THE DRAWINGS

A non-limiting embodiment of the invention will be described by way of example with reference to the accompanying drawings, in which:

Figure 1 shows a side view of an internal combustion engine belt drive featuring a two-arm belt tensioner in accordance with the teachings of the present invention;

Figures 2 and 3 show sections along lines II-II and III-III respectively in Figure 1;

Figure 4 shows a section, with parts removed for clarity, along line IV-IV in Figure 3;

Figure 5 shows a variation of a detail in Figure

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Figure 6 is similar to Figure 2, and shows, with parts removed for clarity, a variation of a detail in Figure 2.

BEST MODE FOR CARRYING OUT THE INVENTION

5       Number 1 in Figure 1 indicates as a whole a belt drive for connecting the output shaft 2 of an internal combustion engine 3 to the shaft 4 of a reversible electric machine 6, i.e. capable of operating as a current generator or motor, and to a shaft 5 of an  
10   auxiliary member 7, e.g. an air-conditioning system compressor.

More specifically, drive 1 comprises a pulley 8 fitted to shaft 2 of the engine; a pulley 9 integral with shaft 4 of the electric machine; and a pulley 10  
15   integral with shaft 5 of auxiliary member 7. Pulleys 8, 9 and 10 are wound with an endless belt 11, which has a first branch 12 extending between pulleys 8 and 9, a second branch 13 extending between pulleys 9 and 10, and a third branch 14 extending between pulleys 8 and  
20   10. Belt 11 is conveniently a poly-V type, and pulleys 8, 9 and 10 each have a corresponding work profile (not shown) with multiple grooves.

With reference to Figure 1, drive 1 also comprises a two-arm belt tensioner 16 cooperating with belt 11 to  
25   ensure correct tensioning of branches 12 and 13 in any operating condition.

Belt tensioner 16 (Figures 1 to 3) comprises a fixed supporting structure 18, in turn comprising a

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curved connecting bracket 19 surrounding the outer casing or frame of electric machine 6, to which it is connected by means of screws, as shown in Figure 1. Bracket 19 is fitted integrally with a cylindrical  
5 tubular body 20 extending coaxially with an axis 21 parallel to the axes of rotation 2a and 4a of shafts 2 and 4. Belt tensioner 16 also comprises a first and a second straight forcing arm 24 and 23, which are connected to supporting structure 18 to rotate in  
10 opposite directions about axis 21, and support respective idle wheels 25, 26 cooperating respectively with branches 12 and 13 of belt 11. Preferably, each arm 23, 24 comprises two contoured portions 27 of the same shape and size and made of pressed sheet metal.  
15 Portions 27 of the same arm face and contact each other on opposite sides of a central plane P of symmetry of relative wheel 25, 26, which plane is perpendicular to the axis A of rotation of the relative wheel. Portions 27 are connected integrally to each other, e.g. welded,  
20 riveted, or by other equivalent joining means, to define, on each arm 23, 24, a fork-shaped end portion 28, to which the relative idle wheel 25, 26 is hinged, and a root or connecting portion 29 separated from the fork-shaped end portion by a straight intermediate  
25 portion 29a. The fork-shaped end portion 28 of each arm 23, 24 has arms 28a, each of which has an integral variable-diameter cylindrical projection 28b, which, when portions 27 are connected to each other, extends

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coaxially with axis A towards the other projection 28b to define, together with the other projection 28b, a hinge pin to which the relative wheel 25, 26 is mounted to rotate about relative axis A via the interposition  
5 of a relative rolling-contact bearing.

Root portion 29 of arm 24 is defined by two adjacent fastening collars 30, whereas root portion 29 of arm 23 is fork-shaped, and comprises two coaxial fastening collars 31 spaced apart sufficiently to  
10 loosely house collars 30 of arm 24 in between.

Root portions 29 of arms 23 and 24 are connected to tubular body 20 by an elastic connecting and forcing device 33 allowing both arms 23 and 24 to rotate about axis 21, and which, in use, pushes arms 23 and 24  
15 towards each other.

As shown particularly in Figures 2 and 3, elastic device 33 comprises a torsionally elastic elongated member 34, which has a constant, conveniently hexagonal, section, extends coaxially with axis 21, and  
20 is defined by a number of parallel, adjacent, elongated bodies 35 (Figures 4 and 5) having respective identical sections in a plane perpendicular to axis 21. Preferably, elongated bodies 35 are cylindrical with a circular section, or prismatic with a triangular,  
25 conveniently equilateral, section. In which latter case, bodies 35 are arranged so that each lateral surface contacts a lateral surface of an adjacent body 35, as shown in Figure 4.

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Elastic member 34 comprises an intermediate portion 36 housed loosely inside tubular body 20; and two opposite end portions 37 and 38 projecting outwards of tubular body 20, and onto which are driven or force-  
5 fitted the outer end portions 39 of respective cylindrical end caps 40 and 41, which define an axial extension of tubular body 20.

More specifically, cap 40 has the same outside diameter as tubular body 20, and comprises a collar-  
10 shaped further end portion 42, opposite relative portion 39, which loosely houses elastic member 34, and in which is firmly connected an end portion of a torsionally rigid, cylindrical tubular sleeve 43 coaxial with axis 21, and whose opposite end portion  
15 43a engages cap 41 in rotary manner about axis 21, and rests against an inner axial shoulder of cap 41. Elastic member 34 extends loosely inside tubular sleeve 43, which loosely engages tubular body 20 and is connected to the tubular body by two bushes 44 of  
20 antifriction material, one of which has an outer radial flange interposed between portion 42 of cap 40 and tubular body 20.

Cap 41 has two end portions of the same outside diameter as cap 40; and an intermediate portion having  
25 two outer radial flanges 45 spaced axially apart and joined by a curved wall 45a. Flanges 45 define respective shoulders, on each of which a relative collar 31 of arm 23 rests and is locked in angularly

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fixed manner. A circumferential slot 46 is formed through the lateral wall of cap 41, between flanges 45, and is fitted through loosely with intermediate portion 29a of arm 24, the collars 30 of which are housed  
5 inside cap 41, coaxially with axis 21, and are fitted to end portion 43a of sleeve 43 in axially and angularly fixed manner, conveniently by a force fit onto an externally splined portion of portion 43a. A cylindrical portion of bush 44 and a further  
10 antifriction bush 48 are interposed between cap 41 and the end portion of sleeve 43; and two rings, also of antifriction material, are interposed between cap 41 and collars 30, and located on opposite axial sides of collars 30 (Figure 2 and 3).

15 In the Figure 6 variation, sleeve 43 is replaced by a further hollow, torsionally elastic member, in this case a wire torsion spring 51, which is connected integrally at opposite ends to cap 40 and arm 24, e.g. with the interposition of a sleeve (not shown) for  
20 supporting arm 24 and connected in rotary and axially fixed manner to tubular body 20. The further elastic member, in this case spring 51, thus defines, with elongated member 34, an elastic assembly defined by two parallel torsionally elastic bodies located one inside  
25 the other.

By providing at least one elastic forcing member coaxial with the common hinge axis of arms 23 and 24, and two straightforward end caps 40 and 41 for



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connecting the opposite ends of the elastic member angularly to arms 23 and 24, the resulting two-arm belt tensioner is therefore much more compact radially than conventional belt tensioners. Moreover, the particular  
5 dimensions of caps 40 and 41 provide for obtaining a belt tensioner, in which tubular body 20, cap 40 and at least part of cap 41 are housed entirely or lie within a straight cylindrical surface K parallel to axis 21, and an intermediate portion of which is defined by the  
10 outer lateral surface of fixed tubular supporting body 20 itself. Obviously, being of constant transverse size along the hinge axis, and in particular of the same size as the tubular supporting body, the range of applications of the two-arm belt tensioner is greatly  
15 increased to also include even particularly compact drives.

Using caps fitted directly to the opposite ends of the elastic member obviously also reduces production cost and, above all, assembly cost and time. Caps 40  
20 and 41, in fact, define respective axial locating and locking stops for both the torsionally elastic member or members 43, 51, and for arms 23 and 24. The caps/torsionally elastic elongated member combination also provides for obtaining a stable assembly in which  
25 the various parts, including the fixed tubular supporting body 20 are packed tightly together by means of a single straightforward axial forcing operation, so that arms 23 and 24 operate in axially fixed positions

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with a constant desired amount of damping. The relative positions of all the various components involved, in fact, are ensured by being determined by respective axial shoulders.

5       Using caps 40 and 41 also provides for immediate, stable connection of arms 23 and 24 to the elastic forcing device. Both the arms, in fact, are simply driven axially onto respective cylindrical portions forming part of the caps directly, or of bodies  
10       connected integrally to the caps.

      Making the elastic member from a bundle of elastic bodies having the same cross section and/or different contacting outer surfaces in relative motion and/or different torsional rigidities, provides for obtaining  
15       an elastic forcing body which is highly elastic, while at the same time capable of exerting the necessary force on the belt to keep the branches taut.

      Striking the right balance between elasticity and applied force is also assisted by the further  
20       torsionally elastic member, e.g. spring 51, which is located compactly parallel to and coaxial with the main elastic member 34, and can be selected from a number of torsional members of different elastic characteristics. The symmetry of the arms provides for action exchange  
25       between the belt and the idle wheels, all lying in plane P; and the fact that the contoured portions of the arms are identical makes the arms modular, so that each can be made using a single die.

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Finally, projections 28b integral with the arms of the fork-shaped end portions enables the wheels to be fitted to the arms with no need for screws, and simultaneously with assembly of the arms.

5        Clearly, changes may be made to belt tensioner 16 as described herein without, however, departing from the scope of the present invention. In particular, arms 23, 24, end caps 40, 41, member 34, and, if provided, further elastic member 51 may be formed otherwise than  
10 as described by way of example. In particular, the arms need not necessarily be curved or symmetrical; and different ways may be provided of connecting caps 40 and 41 to the torsionally elastic member or members, and the arms to the caps.